

# Annularly Grooved Diaphragm Pressure Sensor With Embedded Silicon Nanowires for Low Pressure Application

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**Abstract**—We present a nanoelectromechanical system piezoresistive pressure sensor with annular grooves on the circular diaphragm where silicon nanowires (SiNWs) are embedded as sensing elements around the edge. In comparison with our previous flat diaphragm pressure sensor, this new diaphragm structure enhances the device sensitivity by 2.5 times under pressure range of 0–120 mmHg. By leveraging SiNWs as piezoresistors, this improvement is even remarkable in contrast to other recently reported piezoresistive pressure sensing devices. In addition, with the miniaturized sensing diaphragm (radius of 100  $\mu\text{m}$ ), the sensor can be potentially used as implantable device for low-pressure sensing applications. [2013-0305]

**Index Terms**—Piezoresistance, silicon nanowires (SiNWs), pressure sensor, annular groove, low pressure, biomedical.

## I. INTRODUCTION

**P**IEZORESISTIVE effects in silicon & germanium were early reported in 1950s [1]. Unlike the resistance change caused by the volumetric variation in metallic gauges, the large strain induced resistivity change makes silicon as the promising sensing element in mechanical sensor design [2], [3], which can be easily integrated with standard semiconductor processes [4]–[9]. An enormous amount of research efforts on piezoresistive effects has been devoted in both theoretical [10]–[14] and experimental [3], [15]–[18] ways during past decades. As one of the most sophisticated design utilizing the piezoresistive transduction, pressure sensor based on piezoresistive mechanism has been widely adopted in many areas

diverse from the automotive industry [19], [20] to the biomedical application [21], [22]. Piezoresistive pressure sensor generally demonstrates a lower non-linearity (NL) than that of the capacitive pressure sensing devices under a small diaphragm deflection range [23]. However, such conclusion does not stand in the case of low pressure sensing. For the conventional flat diaphragm based piezoresistive pressure sensor, when the thinner diaphragm with a large deflection is required to sustain the minimum detectable output under a low pressure, the consequent well-known balloon effect will cause a severe degradation in device linearity [24], [25]. In order to solve the problem, a novel boss diaphragm structure, which is configured with a much thicker diaphragm in the center compared to the edge and fabricated based on bulk micromachining, is introduced to concentrate the stress distribution along the beam/rib structure without reducing overall diaphragm thickness and thus remaining a good linearity [25], [26]. Taking the advantage of surface micromachining, Bao *et al.* proposed a modified boss diaphragm by configuring those rib and groove structures on the front side of sensing diaphragm [27], [28] with an improved process accuracy. Further optimization efforts have been made to improve the boss device pressure sensing performance by optimizing geometry factors, such as dimensions, shapes and the ratio between length and width of such rib and groove structures [23], [29], [30]. Besides, the theoretical study has also been carried out to optimize and predict the device sensitivity and linearity [31]–[33]. Nevertheless the low pressure sensing performance of the boss diaphragm pressure sensor has been reaching its inherent limit with the diaphragm area in the range of  $\text{mm}^2$  [23]. A further shrinkage in diaphragm diameter/length will significantly reduce the sensitivity due to a lower resultant resistivity change generated by the deflection induced stress/strain, which is proportional to the square factor of the diaphragm dimension [34]. Hence, other improvement methods have to be developed for a further sensitivity enhancement.

In the past decade, the successful demonstration of implementing nano-scale techniques in both standard CMOS and MEMS technology has been widely reported. Nano-scale materials have been adopted not only for the transistor design [35], [36], but also applied as new sensing element or detection platform [37], [42] for the era of Nano - electromechanical System (NEMS) technology. Among all of promising

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nano-scale sensing elements, both fundamental property and fabrication technology of silicon nanowires (SiNWs) have been well studied over the last decade [43]–[50]. Tremendous experimental efforts have also been carried out for optimizing the performance of nanowires as a piezoresistive sensing element [51], [52]. With the proven stability [53], [54] and excellent piezoresistive sensing performance reported [55], [56], single crystal SiNWs have been successfully implemented into NEMS sensors for various applications (e.g. tactile sensor, mechanical switch and etc.) and resulted in remarkable outcomes [57]–[59].

In this paper, by leveraging the higher concentrated stress profile contributed by annular groove structures on a thin diaphragm (0.5  $\mu\text{m}$  thick in hinge region and 3  $\mu\text{m}$  thick in the rest of the diaphragm area), we successfully demonstrate a new annularly grooved diaphragm pressure sensor using embedded SiNWs as piezoresistors for the low pressure application. The reported proof-of-concept NEMS pressure sensor with miniaturized sensing area, which suits the demands of minimum invasive implantation, can be potentially used for bio-medical applications under a low pressure range.

## II. DESIGN CONSIDERATION

As aforementioned in the introduction, it is difficult to obtain a good low pressure sensing characteristics by simply reducing the diaphragm thickness of the conventional flat diaphragm structure. A balance between excellent sensing resolution and optimum linearity is always the key concern. Thus, the previously reported flat pressure sensor [60] has to be redesigned for the purpose of low pressure detection. Rather than complicated diaphragm structures as reported [23], [27], [29], [33], only the simple structure with annular grooves and ribs around the diaphragm edge is chosen to simplify the fabrication process as well as to prove the concept. The device drawing of our annularly grooved diaphragm pressure sensor is shown in Fig. 1(a) with the detailed groove geometry and diaphragm cross section provided in inset I & II (In Inset II, the top passivation layer ( $\text{Si}_3\text{N}_4$ ) has been turned into transparent for a clearer view). According to the previous study reported by Yasukawa *et al.* [61], the induced surface strain of the both circular boss diaphragm (structure – B) and circular boss diaphragm with rib (structure – B&R) can be expressed as following:

$$\varepsilon_{\text{total}} = \frac{3h_r\zeta}{w_g}\omega + \frac{\pi^2}{16(1+\beta)w_g^2}\omega^2 \quad (1)$$

$$\& \beta = \frac{h_r w_r E_r + h(\pi a E_d - w_r E_r)}{\pi a w_g k_s} \quad (2)$$

Where  $\varepsilon_{\text{total}}$  is the total induced surface strain.  $h_r$  and  $h$  are the thickness of the rib/center boss and hinge region, which is the remaining diaphragm portion below groove structures and connected to the device substrate.  $\omega$  is the diaphragm deflection,  $w_g$  is the width of the groove,  $w_r$  is the width of rib, and  $\zeta$  is the nondimensional coordinate of the gauge position (varies from 1 to -1).  $\beta$  is defined in (2),  $a$  is the radius of the diaphragm,  $E_d$  and  $E_r$  are the equivalent Young's modulus of the diaphragm and rib respectively.  $k_s$  is the support stiffness.

In their study, the total induced strain can be treated as the combination of rib/beam bending strain (the 1<sup>st</sup> portion of the equation (1)) plus the diaphragm deforming strain (the 2<sup>nd</sup> portion of equation (1)). When only structure – B is applied, the equation can be simplified by equal  $h_r$  to  $h$ . Therefore, for structure – B&R, the sensitivity can be further improved by reducing the thickness at the hinge region. In another word, the thickness at the boss region is virtually increased. In order to validate the assumption of a boss structure, the thickness of the center boss region should be at least 6 times of the thickness of the hinge (the remaining thickness at groove region). Additionally, one of initial conditions for the equation of boss structure assumes a negligible bending moment at the center boss region under a given applied pressure. Thus, the mechanical stiffness at the center boss has to be the much higher than that of the hinge region. Such stiffness can be characterized by flexural rigidity as expressed in equation (3).

$$D = \frac{Eh^3}{12(1-\nu^2)} \quad (3)$$

Where  $E$  is the equivalent Young's modulus of the structure and  $h$  is the structure thickness and  $\nu$  is the Poisson's ratio. With a fixed equivalent Young's modulus, the stiffness is highly dependent on the diaphragm thickness. Therefore, the thickness is usually above 10  $\mu\text{m}$  for a good linearity according to the literature [62], especially for those with the center boss structure [23]. In our case, however, the initial diaphragm does not perfectly match the boss configuration (the ratio of the thickness between center and hinge area is only 5.5) and the center thickness may be too thin ( $\sim 3 \mu\text{m}$  as indicated in Fig. 1 Inset II) to be considered as a rigid structure for minimizing center deflections. Nevertheless, the analysis of equation (1) does provide the design guideline for the reconfiguration of original diaphragm structure and to demonstrate the concept of the sensing capability for the SiNWs based NEMS device under a low pressure. The location of piezoresistors (SiNWs in our case) is usually placed along beam/rib region and buried away from the neutral axis. In our design, SiNWs are laid on the BOX layer (buried thermal oxide) and close to bottom surface as shown in Fig. 1, Inset II. In addition, the thickness of rib should be the same as that of the center diaphragm. As indicated in the equation (3), hence, a smaller deflection will be observed at rib region compared to the deflection at hinge region and a resultant lower non-linearity will be accomplished [27].

Besides the sensitivity and linearity improvement for structure – B&R, another advantage of such structure is to minimize the fabrication variation due to the front to back side misalignment. As a result of the front side fabrication, the groove can be patterned with a relatively higher accuracy compared with the result based on backside alignments. Moreover, the well aligned groove re-defines the original stress distribution profile confined by the edge of the conventional flat diaphragm structure, which is released by the back side process. Such stress re-distribution is realized by concentrating the stress along the rib region [29]. Consequently, less performance variations among each individual die is ensured during the final device characterization. The detailed experimental result

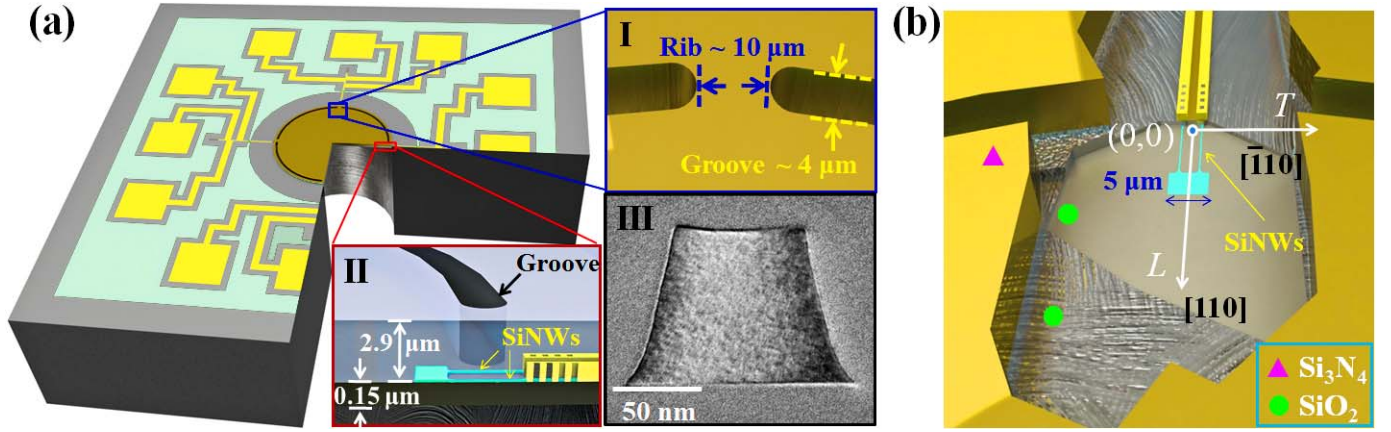


Fig. 1. (a) The schematic of SiNWs embedded pressure with groove and rib structures on the circular sensing diaphragm (released from the back side); (b) The spot view in the location of SiNWs after etching back top passivation layers (2.5 μm Si<sub>3</sub>N<sub>4</sub> and 0.4 μm SiO<sub>2</sub>). SiNWs are patterned along [110] direction as indicated by white arrow. Inset I: the close-up view of the micro-groove; Inset II: the cross-section view of the multilayer diaphragm. Inset III: a TEM image for the cross-section of the nanowire.

for minimizing the device variation will be elucidated in section IV. Additionally, to achieve a better sensitivity and optimum non-linearity, both groove and rib width should be kept as narrow as possible [27], [29]. However, the practical consideration limits the minimization of both groove and rib width in our case. As shown in Fig. 1(b), the width of the paddle connecting two nanowires is around 5 μm. To tolerance alignment errors and etching processes, the width of rib and groove are set to be 10 μm and 4 μm respectively (Shown in Fig. 1 Inset I).

Apart from diaphragm geometry parameters, the nanowire itself also needs to be optimized for better sensing performances. The p-type impurity concentration is firstly designed within the range of 1 to 3 × 10<sup>18</sup> cm<sup>-3</sup>. This is to make sure the optimization of piezoresistive effects, which have to be balanced between a reasonably large piezoresistive coefficient (lower the impurity higher the value) and a relatively acceptable temperature dependency of the piezoresistor (higher the impurity less the temperature induced variation) [12], [16], [18]. It has also been reported that the non-linearity component of piezoresistance is also related to the impurity concentration [25], [31] due to the non-proportional relationship between the splitting of valence band states caused by the energy band shift between heavy and light hole [63]. Details on temperature effects will be discussed in the following chapter. In addition, the cross-section geometry is another important factor to maximize the piezoresistance effect of SiNWs [56]. The ideal cross section view of SiNWs should be a square shape such that the plasma induced damage to the nanowire sidewall can be minimized. However, due to the imperfection of photolithography process, the practical shape of the cross section is always trapezoidal (as shown in Fig. 1 Inset III). Furthermore, the resistance change due to the longitudinal [defined by  $L$  in Fig. 1 (b)] stress component should be maximized and the resistance change caused by transverse [defined by  $T$  in Fig. 1 (b)] stress component has to be kept as minimum for a better overall performance under a given applied normal pressure. The expression of total resistance

changes is shown as below:

$$\frac{\Delta R}{R} = \pi_L \sigma_L + \pi_T \sigma_T \quad (4)$$

Where  $\pi_L$  and  $\pi_T$  are the piezoresistance coefficients along longitudinal and transverse direction, respectively, which for silicon in this orientation have opposite polarities.  $\sigma_L$  and  $\sigma_T$  are the stress components along each direction. As explored by T. Toriyama *et al.* [56], the longitudinal piezoresistance effect can be enhanced by reducing the cross section area of the silicon nanowire. On the other hand, the effect of transverse piezoresistance can be minimized by proper design of the aspect ratio, which is defined as the nanowire thickness divided by its mean width. With the aspect ratio close to 1 (thickness ≈ width), the stress transmission ratio = ( $\sigma_T/\sigma$ ) along the transverse direction can ideally approximate to 1, therefore, maximizing the transformation of stress component in the longitudinal direction. Here,  $\sigma$  is the normal stress applied to the substrate. Meanwhile, the change of aspect ratio does not affect the stress component along the longitudinal direction. The detailed process for the optimization of the SiNWs cross section geometry will be described in the next chapter.

### III. DEVICE FABRICATION

As shown in Fig. 2(a), the device fabrication starts on (100) plane single crystal SOI wafer with the device layer of 117 nm and BOX layer (buried thermal oxide) of 145 nm. The global implantation with a boron dosage of 1 × 10<sup>14</sup> ion/cm<sup>2</sup> is conducted followed by a 30 seconds rapid thermal annealing (RTA) process at 1050 °C for dopant activation. The resultant final impurity concentration of SiNWs is around 3.5 × 10<sup>18</sup> cm<sup>-3</sup> [64]. The focus exposure matrix process is carried out on test wafers to optimize the exposure recipe prior to the photolithography process. The first photolithography is then performed using the stepper mask (feature projection) based on the result from the focus exposure matrix with the critical dimension ~150 nm after developments. The photoresist is then trimmed for 60 seconds by plasma-induced feeding gas

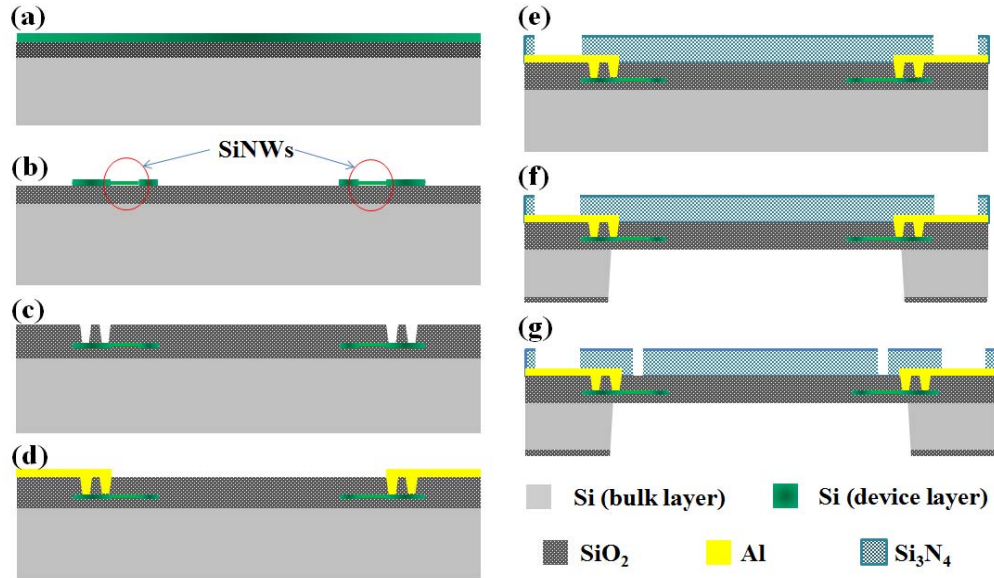


Fig. 2. Device process flows of SiNWs embedded pressure sensor with grooves on the front diaphragm. (a) the starting (100) plane SOI wafer; (b) the formation of SiNWs along  $\langle 110 \rangle$  direction; (c)-(d) Deposition of 1<sup>st</sup> passivation layer (400 nm PECVD  $\text{SiO}_2$ ) and metallization; (e) Deposition of 2<sup>nd</sup> passivation layer (2.5  $\mu\text{m}$  PECVD  $\text{Si}_3\text{N}_4$ ) and bond pad opening; (f) backside DRIE for the diaphragm release; (g) the pattern of grooves by etching away  $\text{Si}_3\text{N}_4$  on the front side diaphragm.

( $\text{He}/\text{O}_2 + \text{N}_2$ ) and result in the minimum feature size of 110 nm in final. The plasma etching is processed to form the SiNWs along  $\langle 110 \rangle$  direction as shown in Fig. 2(b). This orientation of p-type SiNWs on (100) plane [65]. With further thermal dry oxidation at 875 °C for 2 hours, SiNWs with an average cross section of 90 nm  $\times$  90 nm (resultant aspect ratio  $\approx 1$  as shown in inset III of Fig. 1) and various lengths (1  $\mu\text{m}$ , 2  $\mu\text{m}$ , 5  $\mu\text{m}$  and 10  $\mu\text{m}$ ) are fabricated. The 2<sup>nd</sup> implantation with a boron dosage of  $1 \times 10^{15}$  ion/ $\text{cm}^2$  is conducted at only the contact and paddle region. After deposition of 1<sup>st</sup> passivation layer (400 nm of  $\text{SiO}_2$ ), via is drilled through top  $\text{SiO}_2$  layer to the bottom highly doped silicon layer [Fig. 2(c)]. A 750 nm Al plus another 25 nm TaN layer (serving as adhesion layer between Si and Al) are sputtered and patterned to form the electrical connection [Fig. 2(d)]. Sequentially, A 2.5  $\mu\text{m}$  thick low stress ( $\sim 84$  MPa)  $\text{Si}_3\text{N}_4$  is then coated by using plasma-enhanced chemical vapor deposition (PECVD) to compensate the residual stress from beneath oxide layers. As a result, the flat diaphragm with initial deflection less than 100 nm is targeted [60]. After bond pad opening shown in Fig. 2(e), the front side sensing diaphragm is successfully defined through the back-side deep reactive-ion etch (DRIE) with the BOX layer (buried thermal oxide) as an etching stopper shown in Fig. 2(f). However, due to the non-uniformity of inductively coupled plasma etching, the deviation of final released diaphragm is as large as 12 % [as shown in Fig. 7(a)] from the target diaphragm (diameter of 200  $\mu\text{m}$ ), especially at the center and edge ring shaped area around the 8-inch wafer. Finally the front side annular groove is defined with a contact glass mask and patterned through the etching process on the 2.5  $\mu\text{m}$   $\text{Si}_3\text{N}_4$  cladding layer in an annular region along the diaphragm edge. The beneath  $\text{SiO}_2$  layer ( $\sim 500$  nm thick) is used as the etching stopper [Fig. 2(g)].

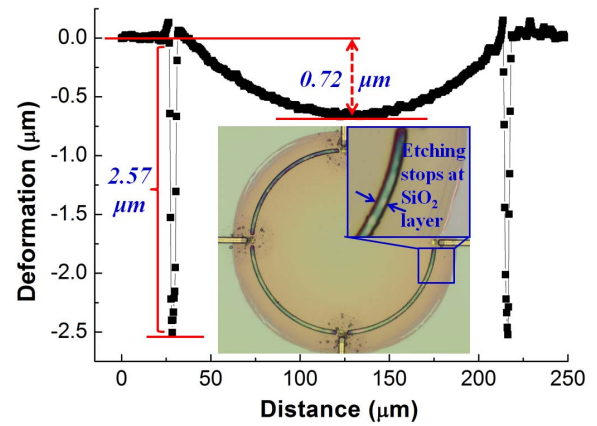


Fig. 3. The plot for the initial deflection of the annularly grooved diaphragm. The diaphragm deforms downward and forms a concave shape. Inset shows the optical image of the pressure diaphragm and a zoom-in view of the groove.

#### IV. EXPERIMENT RESULT AND DISCUSSION

##### A. Sensitivity and Linearity

The optical image of the release diaphragm with groove and rib structures is given in the inset of Fig. 3. The zoom-in image on the groove region indicates a successful etching stop on beneath  $\text{SiO}_2$  layers and this is further verified by capturing the diaphragm topography using white light interferometer (WYKO NT3300). With the depth of groove about 2.57  $\mu\text{m}$ , the over etch of  $\text{SiO}_2$  layer ( $\sim 70$  nm) is about 12% of the total thickness of  $\text{SiO}_2$  layers (400 nm PECVD  $\text{SiO}_2$  layer plus 145 nm thermal  $\text{SiO}_2$  layer). However, it is also plotted as the trade-off of a significantly reduction in the hinge region (thus the reduction of the flexural rigidity), the initial diaphragm deflection of 0.72  $\mu\text{m}$  is found and it is about a quarter of the total diaphragm thickness ( $\sim 3$   $\mu\text{m}$ ). To further understand the initial diaphragm deflection, the simulation on



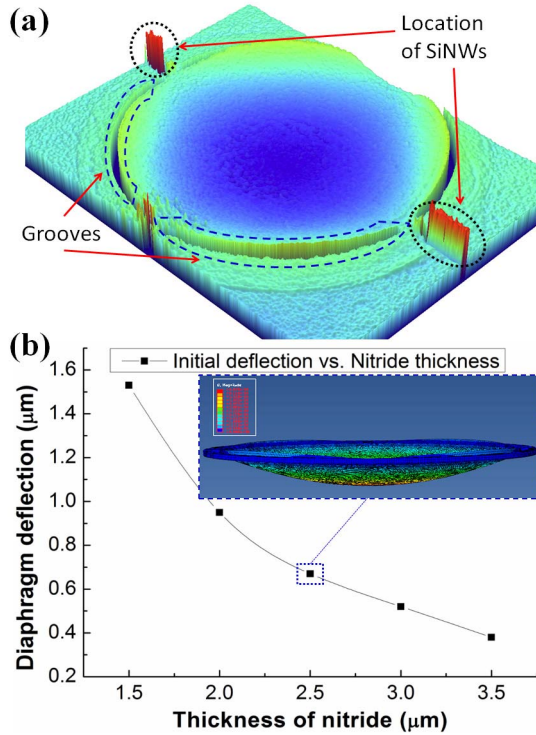


Fig. 4. (a) The surface profile captured by Wyko white light system and (b) the simulated relationship between the nitride layer thickness and the initial diaphragm deflection. The inset shows the simulated diaphragm shape (concave) when a 2.5 μm thick nitride is used as the cladding layer.

the effect of residual stresses contributed by the multi-layered diaphragm (named in the sequence from bottom to top: buried thermal oxide layer, PECVD oxide layer and PECVD low stress nitride layer) is conducted. The simulation is conducted by using Abaqus 6.10 and all parameters used in simulation are listed in Table I. The result of diaphragm deflections with respect to different thicknesses of nitride layer is plotted in Fig. 4(b) and it reveals the inverse proportionality between the diaphragm deflection and the cladding nitride layer thickness. In addition, both the simulation result [inset of Fig. 4(b)] and the measurement data [Fig. 4(a)] indicate a concave shape diaphragm. The value of the simulated deflection ( $\sim 0.67 \mu\text{m}$ ) is very close to the experimental data ( $\sim 0.72 \mu\text{m}$ ) captured by Wyko white light system. As a rule of thumb, a deflection less than 10 % of the total diaphragm thickness will introduce a mechanical non-linearity of 0.2 % and a deflection less than 30 % of the total thickness will cause a non-linearity component as large as 2 %. For a deflection large than 30% of the diaphragm thickness, this assumption of the small deflection principle is no longer valid [66].

To verify the performance difference of the SiNWs based pressure sensor with the new annularly grooved diaphragm in contrast to the previously reported flat diaphragm counterpart, the percentage of resistance changes over the initial resistance (when no pressure is applied) are measured in the form of  $\Delta R/R$  (%) by varying pressure changes on X-axis. The testing is firstly conducted under room temperature (25 °C) with supply voltage of 0.5 V. The resistance change is measured by the semiconductor characterization

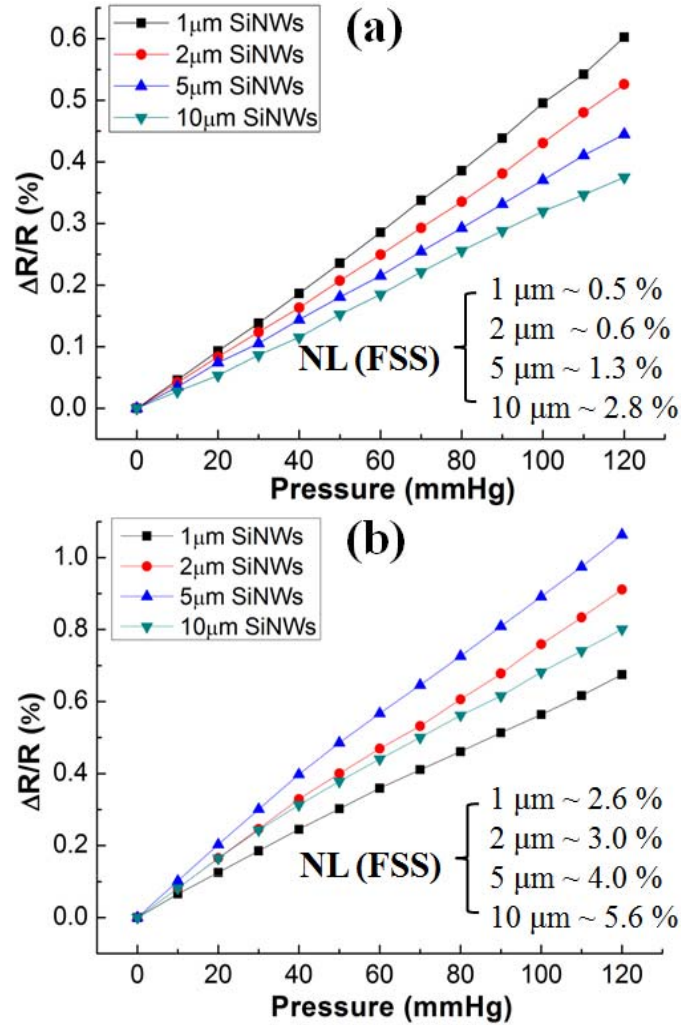


Fig. 5. Plots of the percentage changes of resistance ( $\Delta R/R$  (%)) with respect to pressure changes for (a) the conventional flat diaphragm SiNWs pressure sensor and (b) the annularly grooved diaphragm SiNWs pressure sensor. The calculated nonlinearity for pressure sensors with different lengths of SiNWs (1 μm, 2 μm, 5 μm and 10 μm) is given in insets.

TABLE I  
MATERIAL PROPERTIES APPLIED IN THE  
FINITE ELEMENT METHOD (FEM)

Material	Young's modulus	Poisson's ratio	Residual stress (MPa)	Density (g/cm <sup>3</sup> )
Si <sub>3</sub> N <sub>4</sub>	170 GPa	0.27	85 (tensile)	3.44
SiO <sub>2</sub>	60 GPa	0.20	-250 (compressive)	2.65

system (Keithley 4200-SCS) with the compressed air source applied by the pressure regulator (ALICAT PCD Series) from the backside of the device (please refer to [60] for more detailed testing setups). As a result, piezoresistors will experience a uniform tensile stress. Fig. 5(a) illustrates the resistance change of the traditional flat diaphragm pressure sensor, the maximum average percentage change ( $\sim 0.6$  %)

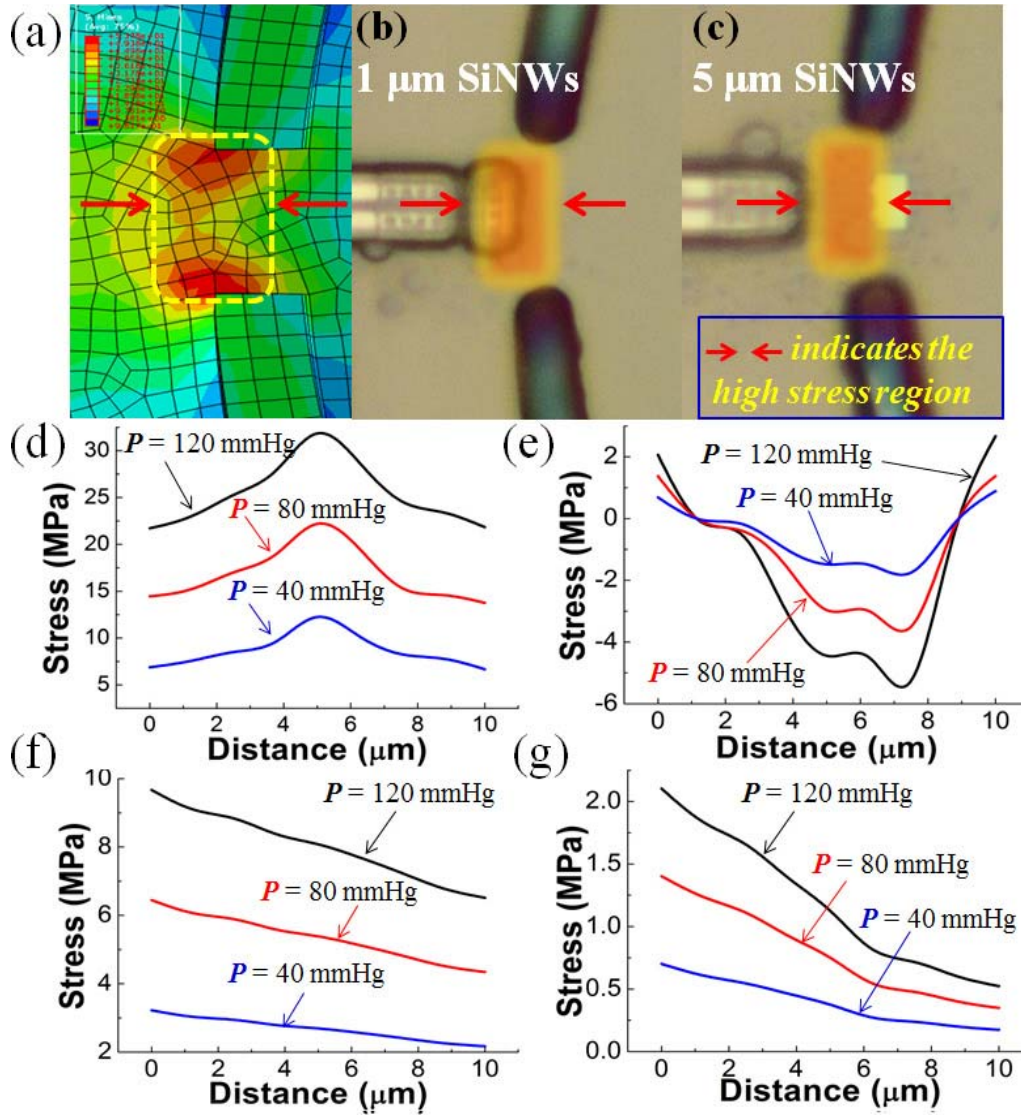


Fig. 6. (a) Illustration of high stress region captured from FEM, (b) - (c) zoom-in optical images of rib region for 1 and 5  $\mu\text{m}$  SiNWs designs respectively. Simulation results of (d) - (e) extracted stress distributions for both longitudinal and transverse stress component along  $L$ -direction of the grooved diaphragm pressure sensor, (f) - (g) extracted stress distributions for both longitudinal and transverse stress component along  $L$ -direction of the conventional flat diaphragm pressure sensor.

occurs for the 1  $\mu\text{m}$  SiNWs embedded pressure sensor among other designs with longer length of embedded SiNWs. The reasonable explanation has been addressed based on changes of stress distribution profiles along the radial direction of the diaphragm [60]. When SiNWs are located away from the diaphragm edge, the average longitudinal stress experienced along the nanowires gradually decays as the stress distribution profile shown in Fig. 6(f). In the case of the grooved diaphragm pressure sensor, the maximum average percentage change of 1.07 % for 5  $\mu\text{m}$  SiNWs pressure sensor is observed [Fig. 5(b)]. In contrast to the 5  $\mu\text{m}$  SiNWs embedded flat diaphragm pressure sensor, the improvement of percentage changes is about 2.5 times and this change is in a good agreement with the finding of the increment of the average stress distribution along longitudinal direction from the simulation.

The non-linearity ( $NL$ ) for both the flat diaphragm and the grooved diaphragm pressure sensor with different

lengths of SiNWs over full-scale span (FSS) are also calculated based on equation (5) [63] and provided in insets.

$$NL = \frac{R(T) - \{[R(T_m) - R(0)](T/T_m) + R(0)\}}{R(T_m) - R(0)} \quad (5)$$

Where,  $R(0)$  &  $R(T)$  are output resistance values at the initial condition (pressure = 0) and a given pressure status (pressure =  $T$ ), respectively. The pressure varies from 0 to  $T_m$  (maximum pressure applied). As predicted from the initial deflection profile plotted in Fig. 3, a larger non-linearity is introduced after reconfigurations of the original flat diaphragm structure. Beside an extra increment of non-linearity, an obvious sensitivity shift from the previously reported flat diaphragm with 1  $\mu\text{m}$  SiNWs to the currently reported grooved diaphragm with 5  $\mu\text{m}$  SiNWs is observed. To understand this shift, the FEM is conducted and zoom-in views of the

beam/rib structure are also provided in Fig. 6(b) and (c) with a reference image showing the high stress distribution region from FEM [Fig. 6(a)]. As depicted in Fig. 6(a), the highest stress region is located slightly behind (around  $2\ \mu\text{m}$  behind) where groove structures are fabricated along the rib. The location of  $1\ \mu\text{m}$  SiNWs [shown in Fig. 6(b)] is out of the high stress region, whereas, 75~80% portion of  $5\ \mu\text{m}$  nanowire [shown in Fig. 6(c)] is completely merged within such high stress region. The detailed stress distribution profiles along the rib [ $L$ -direction defined in Fig. 1(b)] under three different external pressures are plotted in Fig. 6(d). The point  $(0, 0)$  [indicated in Fig. 1(b)] is defined as the origin for the  $X$ -axis (distance). With the same explanation, the average stress distributed along  $2\ \mu\text{m}$  SiNWs is higher than that of  $1\ \mu\text{m}$  SiNWs design but lower than the average stress of  $5\ \mu\text{m}$  SiNWs design. For the  $10\ \mu\text{m}$  SiNWs, however, it is too long to be entirely confined within the high stress region, thus, experiencing a lower average stress. In addition, the transverse stress distribution profile with the same origin, span and direction are plotted in Fig. 6(e) as well. Unlike the linearly decayed stress distribution from edge (tensile stress) to the center (compressive stress) in the case of flat diaphragm pressure sensor [indicated in Fig. 6(g)], it is worth noting that the stress distribution profile is no longer linear after fabrication with grooves. The stress profile ramps from the small tensile down to relatively larger compressive region and reaches its maximum (compressive) at distance about  $7\ \mu\text{m}$  away from the origin. It then ramps up again to tensile stress region. Although this stress change does not dominate the difference in the final resistance value, it theoretically reduces the average transverse resistance changes by approximate 10%, if the same length ( $7\ \mu\text{m}$ ) of the nanowires is allocated within exact the compressive stress region. Hence it leads to improvement of the total resistance change as defined in equation (4). However, the larger transverse stress will also cause an increased non-linearity component regardless of the stress polarity (both compressive and tensile) [63], therefore, such compressive stress profile may also provide a negative contribution to the device linearity. To further understand the non-linearity component, a more closed-form expression of non-linearity between an applied pressure and the final resistance change of a given piezoresistor can be described as the following [61]:

$$NL_{p-R} = NL_{\varepsilon-R} + NL_{\omega-\varepsilon} + NL_{p-\omega} \quad (6)$$

Where  $NL_{p-R}$  is the total non-linearity component between applied pressure and final resistance changes;  $NL_{\varepsilon-R}$  is non-linearity component introduced between strain/stress and resistance changes;  $NL_{\omega-\varepsilon}$  is non-linearity component introduced between deflection and strain changes;  $NL_{p-\omega}$  is non-linearity component introduced between applied pressure and mechanical deflection changes. We have examined non-linearity components contributed by both  $NL_{p-\omega}$  ( $\sim 2\%$  due to the initial diaphragm deflection) and  $NL_{\varepsilon-R}$ . The  $NL_{\omega-\varepsilon}$  is usually small and can be removed by an external circuit. For instance, by applying the Wheatstone bridge structure,  $NL_{\omega-\varepsilon}$  can be further minimized [61].

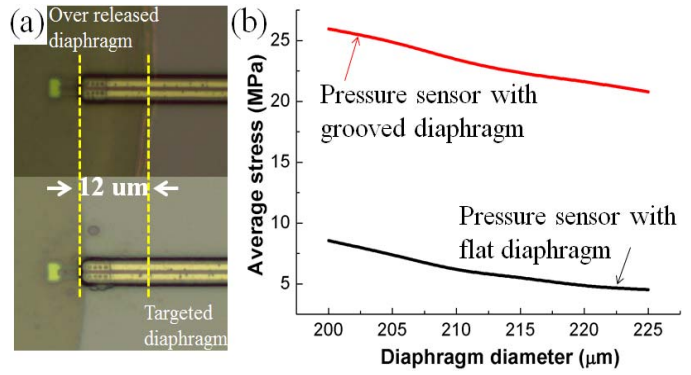


Fig. 7. (a) Optical images of over-release diaphragm (top) and targeted diaphragm (bottom) with embedded  $5\ \mu\text{m}$  SiNWs, (b) Simulation results of extracted average stress along  $5\ \mu\text{m}$  SiNWs for both grooved and flat diaphragm pressure sensor with respect to the diaphragm diameter variation.

### B. Improvement on Process Variations

Another aforementioned advantage of the groove structure is the reconfiguration of the stress distribution. As illustrated in Fig. 7(a), the backside release process has introduced large fabrication variations and the over-release of the diaphragm can be up to 12% of the targeted dimension (diameter of  $200\ \mu\text{m}$ ). As a consequence, the performance of the released device varies significantly from die to die. The average stress distributed along  $5\ \mu\text{m}$  SiNWs with respect to the diaphragm size variation for both the flat and the grooved diaphragm pressure sensor is extracted by using FEM [plotted in Fig. 7(b)]. For a flat diaphragm structure, the amount of average stress drops more than 45 % by varying the diaphragm diameter from  $200\ \mu\text{m}$  up to  $225\ \mu\text{m}$ . In case of the grooved diaphragm, such diameter variation affects the average stress change up to only 18 %. The FEM is conducted based on a linear perturbation model. For further verification, the experiment is conducted on different pressure sensor samples ( $n = 10$ ) with various diaphragm diameters. Output resistance variations for  $5\ \mu\text{m}$  SiNWs embedded in both flat and groove diaphragm pressure sensor are shown in Fig. 8. The blue curve with error bars reflects results for the flat diaphragm pressure sensor (referring to bottom  $X$  and left  $Y$ -axis) with the deviation of resistance changes up to 33% of its mean value at pressure of  $120\ \text{mmHg}$ . On the other hand, as plotted in red curve with error bars (referring to top  $X$  and right  $Y$ -axis), the variation of diaphragm size only introduces 14 % of performance differences to the annularly grooved pressure sensor within the same pressure range. The obvious improvement of the device sensing performance over process variations is hereby reported and it is contributed by the groove structure, which forces the stress distribution along the rib.

Here, the reported device is just the proof-of-concept for the sensing capability of SiNWs embedded sensor under a low pressure and the design has not been completely optimized yet. Further sensitivity improvement can be realized by changing circular diaphragm to a square shape. This change may lead to the sensitivity improvement up to 60% base on flat plate design theory [66]. Additionally, the optimum location of SiNWs with respect to the annular groove has not been determined.



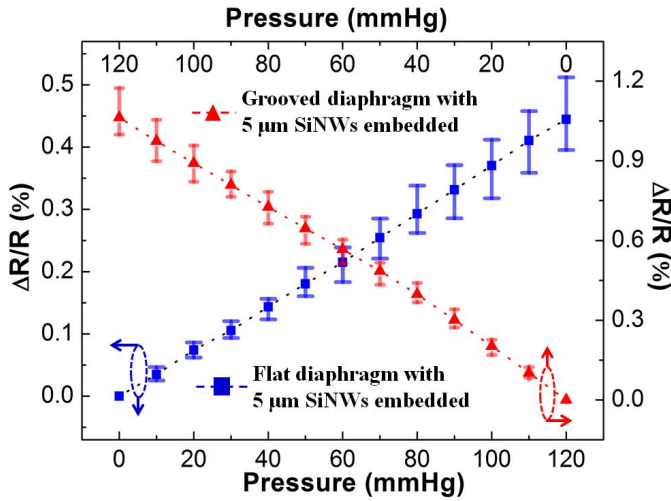


Fig. 8. Plot of output resistance variations with the blue curve and error bars for the flat diaphragm pressure sensor (refer to bottom X-and left Y-axis) and the red curve and error bars (refer to top X-and right Y-axis) for the grooved diaphragm pressure sensor with 5  $\mu\text{m}$  SiNWs embedded in both cases.

The location of 1  $\mu\text{m}$  SiNWs can be re-adjusted to completely within the high stress region. This will further improve the piezoresistance changes by  $\sim 40\%$  due to the reported higher gauge factor ( $G$ ) of 1  $\mu\text{m}$  SiNWs ( $G \sim 100$ ) over that of 5  $\mu\text{m}$  design ( $G \sim 65$ ) [60]. The orientation of shorter SiNWs can also be re-aligned along the tangential direction [ $T$ -direction as defined in Fig. 1 (b)] without increasing the width of rib. As a result, further sensitivity enhancement will be achieved due to a relatively uniform average stress along the tangential direction at the narrow rib. In addition, non-linearity components will also be compensated among piezoresistors, when they are subjected to lateral stress [67]. Furthermore, the dimension and thickness of both rib and groove can be refined for a higher stress distribution profile and lower non-linearity.

### C. Temperature Effects

Temperature variation is another key factor, which affects several parameters like material property, feature geometry and mostly importantly the piezoresistive effect [2], [11], [12]. The contribution from other temperature dependent factors varies and is relatively smaller compared with that from the piezoresistor itself [30]. For the p-type piezoresistor, the reported experimental result reveals the dependency of shear piezoresistance coefficient on both impurity concentrations and temperature variations [16], [18]. A more general expression for the relationship between the piezoresistance and its temperature dependency is defined by Y. Kanda [65] as:

$$\Pi(N, T) = P(N, T) * \Pi(300) \quad (7)$$

Where  $\Pi(N, T)$  is the piezoresistance coefficient with an impurity concentration  $N$  at a temperature  $T$ .  $\Pi(300)$  is the piezoresistance coefficient at temperature of 300 K.  $P(N, T)$  is the piezoresistance factor and can be expressed as:

$$P(N, T) = \frac{300}{T \ln(1 + e^{E_f/kT})(1 + e^{-E_f/kT})} \quad (8)$$

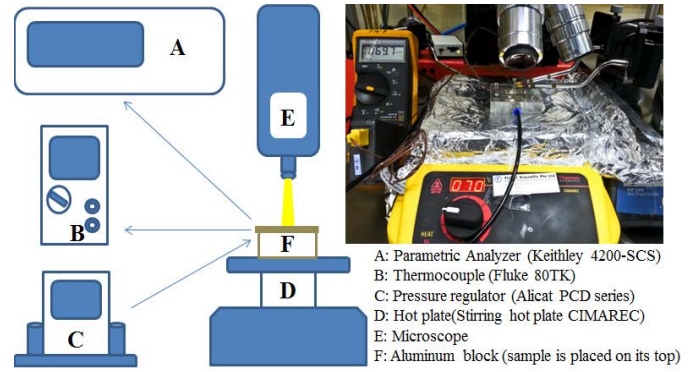


Fig. 9. Illustration of setups for the temperature response test of the reported pressure sensor.

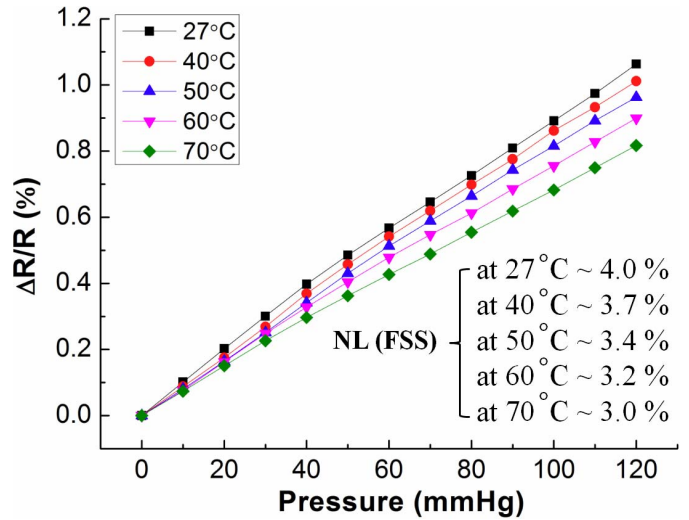


Fig. 10. Output resistance changes under different ambient temperatures (27 °C to 70 °C). The plotted data is measured from reported groove diaphragm pressure sensors with 5  $\mu\text{m}$  long SiNWs as piezoresistors. Insets show the non-linearity (NL) component with respect to outputs at different temperatures.

Where, the  $E_f$  is the Fermi energy and it is related to the impurity concentration. It has been reported that equation (7) may be suitable for the first-order approximation and the simulated result matches the experimental data in case of pure circular and square diaphragm based pressure sensors [30]. In order to explore the temperature response of our device, the hot plate is used as the heat source. Temperature changes are monitored by thermocouple with accuracy around  $\pm 2^\circ\text{C}$  of its display value. The device is assembled on top of an aluminum block by an acrylic plate. The gasket is used to surround the testing sample for air sealing purpose. The air pressure is applied from the backside of the test sample by a pressure regulator (The similar Al sealing block has also been reported in [64]). The electrical path is established by probing bonding pads through a small opening window on the acrylic plate and resistance changes are recorded by the parametric analyzer. The detailed setup is shown in Fig. 9.

The response of the device is examined within temperatures varying from 27 °C (room temperature) to 70 °C, which is a reasonable temperature range for most bio-medical related



TABLE II  
THE COMPARISON OF BIO-MEDICAL PIEZORESISTIVE PRESSURE SENSING DEVICES BETWEEN THIS WORK AND OTHER REPORTED DESIGN

Ref.	Applications of sensing	Sensing element	Effective sensing area ( $\mu\text{m}^2$ )	Sensitivity $((\Delta R/R)/\text{mmHg})$	Normalized Sensitivity $((\Delta R/R)/\text{mmHg}/\mu\text{m}^2)$
[68]	Intracranial pressure	P-doped polysilicon wires	$\pi * 500^2 = 785398$	$\sim 13 \times 10^{-5}$	$\sim 1.66 \times 10^{-10}$
[10]	Cardiovascular blood pressure	P-doped silicon wires	$600 * 600 = 360000$	$\sim 4 \times 10^{-5}$	$\sim 1.1 \times 10^{-10}$
[9]	Coronary blood pressure	Polysilicon strain gauge	$100 * 100 = 10000$	$\sim 0.07 \times 10^{-5}$	$\sim 0.7 \times 10^{-10}$
[69]	Intraocular pressure	NiCr	$5400 * 5400 = 29160000$	$\sim 80.8 \times 10^{-5}$	$\sim 0.28 \times 10^{-10}$
Previous work [60] (1 $\mu\text{m}$ SiNWs)	Other application	P-doped SiNWs	$\pi * 100^2 = 31416$	$\sim 5 \times 10^{-5}$	$\sim 15.9 \times 10^{-10}$
This work (5 $\mu\text{m}$ SiNWs)	low pressure for bio-medical purpose	P-doped SiNWs	$\pi * 100^2 = 31416$	$\sim 8.92 \times 10^{-5}$	$\sim 28.4 \times 10^{-10}$

applications. The tip portion of the thermocouple is closely attached on the top surface of the aluminum block (just beside the test sample). Once the reading from the thermocouple is stabilized, another 20 minutes interval is spent before tightening the acrylic plate by screws. This minimizes any testing errors introduced by the setup. Additionally, the entire hot plate is covered with aluminum foils, which is connected to the common ground, to reduce the noise signal generated from the testing instrument. The temperature testing result of the grooved pressure sensor with 5  $\mu\text{m}$  SiNWs is plotted in Fig. 10. The obvious sensitivity drop over temperature increment is reported. Compared with the output resistance change at room temperature (27 °C), this drop of output performance is as large as 23% at 70 °C. Contrary to the sensitivity degradation, the linearity component improves at higher temperature. Based on equation (5), the non-linearity under the full scale span is around 3% (shown in the set of Fig. 10) at 70 °C. Such temperature dependent linearity change is in a good agreement with the literature [25].

## V. CONCLUSION

To provide a clearer view for the performance of the grooved diaphragm based SiNWs sensing device under a low pressure, especially for bio-medical applications, a comparison is tabulated in Table II with all references reported for different bio-medical applications except our previous work. It is shown in Table II that our reported SiNWs based devices demonstrate the great scalability (indicated in column 4). The sensitivity  $((\Delta R/R)/\text{mmHg})$  for all devices is also summarized in column 5. It seems that the performance is highly dependent on the area of sensing diaphragm. In fact, the relationship between the effective sensing area and the device sensitivity should be linearly proportional as predicted by the plate theory [34]. Thus, a fair sensitivity comparison among all types of pressure sensing devices is listed in the last column of the table. After normalizations of variations in the effective sensing

area, the sensitivity improvement of the previously reported flat diaphragm based SiNWs pressure sensor is almost an order over other literature reports. With a further improvement benefited by the groove structure, the currently reported device boosts up the sensitivity by at least 17 times compared with results reported by other groups. Such significant improvement proves the feasibility for implementing SiNWs based device for low pressure sensing applications by simply reconfiguring the device diaphragm.

In summary, the new annularly grooved diaphragm pressure sensor with SiNWs embedded as a piezoresistor is reported here. Various design considerations for both sensitivity improvement and minimization of nonlinearity have been discussed in detail. Experimental results are summarized and analyzed with respect to results from the previous flat diaphragm based SiNWs pressure sensor. Both FEM and measurement data explain the benefit of the groove structure that forces the stress to be concentrated around the rib region and results in a greater resistance change. The performance variation of the device over a reasonable temperature range is examined. A table is also summarized with an explicit indication for the enhancement of SiNWs based devices over other reported traditional piezoresistive pressure sensors. Benefiting by the superiority of SiNWs, the reported proof-of-concept device with the groove diaphragm structure further enhances the sensing capability and fulfills the demand for working under the low pressure range required by implantable bio-medical applications.

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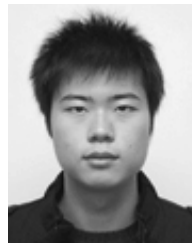
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